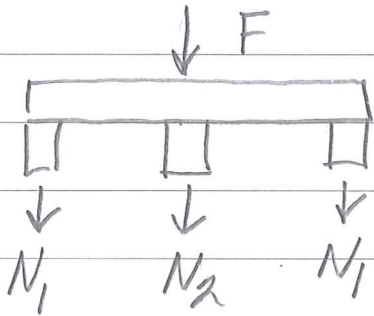


Z2 Hållfasthetslära TMED17 2015-04-14

1) Ställ upp jämvikt (Antag F så stor att det blir kontakt vid B)



$$\uparrow -2N_1 - N_2 - F = 0 \quad 1)$$

$$\text{eller } 2\sigma_1 A_1 + \sigma_2 A_2 = -F \quad 1)$$

deformationssamband, centrisk last F

$$\delta_1 = \delta_2 + \Delta \quad 2) \quad \Delta = \text{gap} = 0.1 \text{ mm}$$

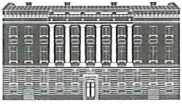
materialsamband, LB p 9-11

$$\epsilon_1 = \frac{\sigma_1}{E_1} = \frac{\delta_1}{L} \quad 3) \quad \epsilon_2 = \frac{\sigma_2}{E_2} = \frac{\delta_2}{L} \quad 4)$$

$$3), 4) \text{ i } 2) \Rightarrow \frac{\sigma_1 L}{E_1} = \frac{\sigma_2 L}{E_2} + \Delta \Rightarrow$$

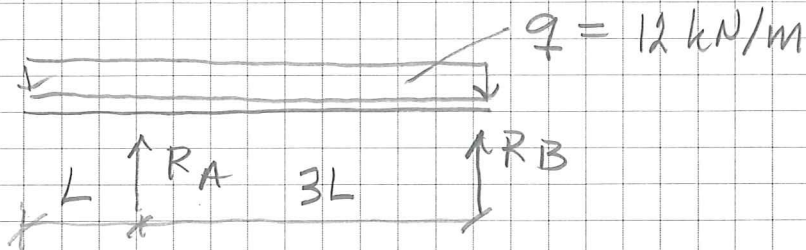
$$\sigma_1 = \sigma_2 \frac{E_1}{E_2} + \frac{\Delta E_1}{L} \quad \text{i } 1) \Rightarrow$$

$$\sigma_2 = \frac{-F - \frac{2\Delta E_1 A_1}{L}}{2A_1 \frac{E_1}{E_2} + A_2} = -240 \text{ MPa}$$



22 Hållfasthetslära TME017 2015-04-14

2) Bestäm stödreaktioner:



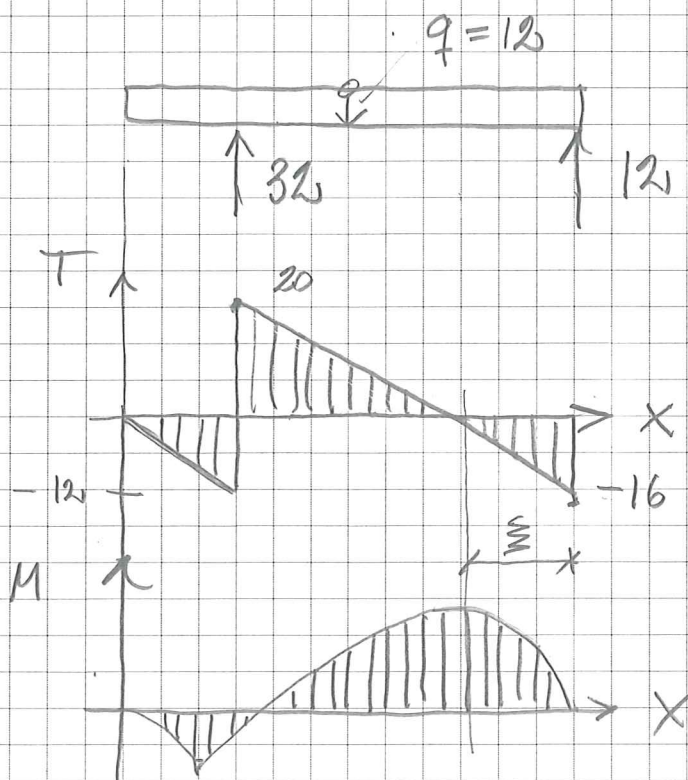
$$\begin{aligned} \uparrow \cdot \quad R_A + R_B - q \cdot 4L &= 0 \\ \curvearrowright_B \quad q \cdot 4L \cdot 2L - R_A \cdot 3L &= 0 \end{aligned} \Rightarrow \begin{aligned} R_A &= \frac{8}{3} qL = 32 \text{ kN} \\ R_B &= \frac{4}{3} qL = 16 \text{ kN} \end{aligned}$$

Rita tvärkrafts momentdiagram

Använd LB p 70

$$\frac{dT}{dx} = -q \rightarrow T(x_2) - T(x_1) = - \int_{x_1}^{x_2} q(x) dx$$

$$\frac{dM}{dx} = T \rightarrow M(x_2) - M(x_1) = \int_{x_1}^{x_2} T(x) dx$$



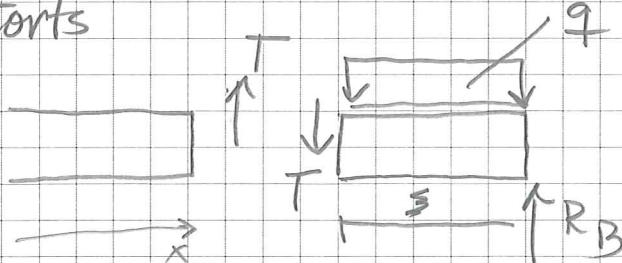
$$T_{\max} = 20 \text{ kN}$$

Hitta M_{\max}



7.2 Hållfasthetslära TME017 2015-04-14

2) Forts



$$\uparrow -T - q \cdot \xi + R_B = 0$$

$$T(\xi) = R_B - q \xi$$

$$T(\xi) = 0 \rightarrow \xi = \frac{R_B}{q} = \frac{4}{3} L = 1.33 \text{ m}$$

$$\rightarrow M(4L) - M(\xi) = \int_{\xi}^{4L} T(x) dx = \frac{1}{2} (-16) \cdot 1.33 = -10.64 \text{ kNm}$$

För rektangulärt tvärsnitt $I_y = \frac{bh^3}{12}$ $A = bh$

LB p 80-82 $\Rightarrow \sigma_{\max} = \frac{M_{\max}}{W_b} = \frac{M_{\max} \cdot 6}{bh^2}$

LB p 89 $\Rightarrow \tau_{\max} = \frac{3}{2} \frac{T_{\max}}{A} = \frac{3}{2} \frac{T_{\max}}{bh}$

har är $h = 1.5b$ $\sigma_{\text{till}} = 9 \text{ MPa}$ $\tau_{\text{till}} = 0.6 \text{ MPa}$

varmed $\sigma_{\text{till}} \geq \frac{M_{\max} \cdot 6}{b(1.5b)^2} \rightarrow b \leq \sqrt[3]{\frac{6 M_{\max}}{2.25 \sigma_{\text{till}}}} = 147 \text{ mm}$

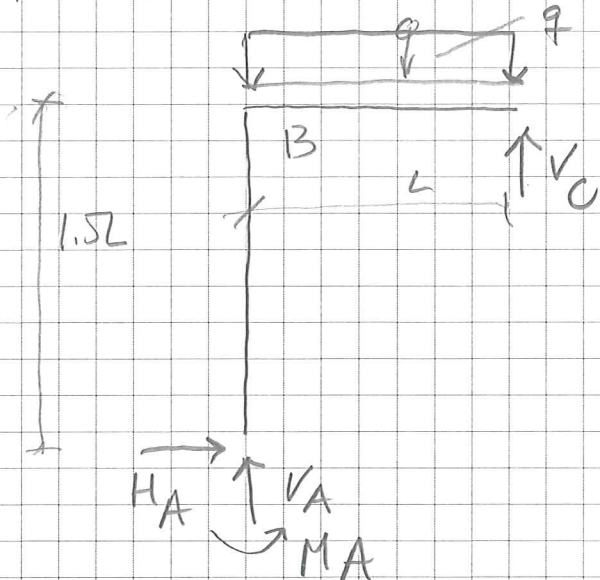
$\tau_{\text{till}} \geq \frac{T_{\max} \cdot 3}{2b \cdot 1.5b} \rightarrow b \leq \sqrt{\frac{T_{\max}}{\tau_{\text{till}}}} = 183 \text{ mm}$

\rightarrow slyjvning dimensionerande



Z2 Hållfasthetslära TMED17 2015-04-14

3) Inlör stödreaktioner



Jämvikt för frigjord ram

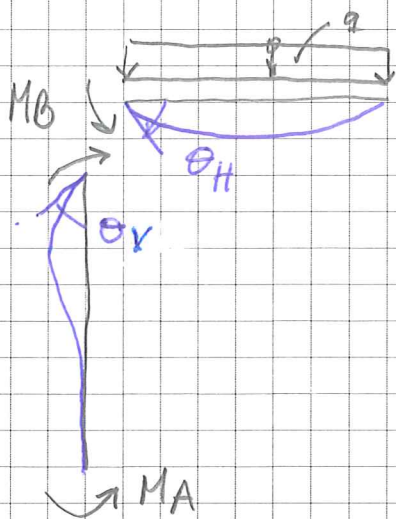
$$\rightarrow H_A = 0 \quad 1)$$

$$\uparrow V_A + V_C - Q = 0 \quad 2)$$

$$\curvearrowright H_A \cdot 1.5L + V_C \cdot L + M_A - \frac{QL}{2} = 0 \quad 3)$$

Bestäm M_A och moment vid B

se deformationssamband (Fs 6.3 & 6.5)



Här gäller $\theta_H = \theta_V$ (LBP)

$$\theta_H = \frac{QL^2}{24EI} - \frac{M_B L}{3EI}$$

$$\theta_V = \frac{M_B \cdot 3L}{2 \cdot 4EI}$$

$$\rightarrow \frac{QL}{24} = M_B \left(\frac{3}{8} + \frac{1}{3} \right) \Rightarrow M_B = \frac{QL}{17}$$

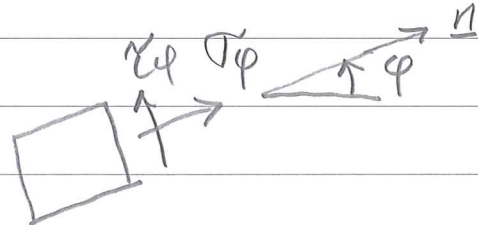
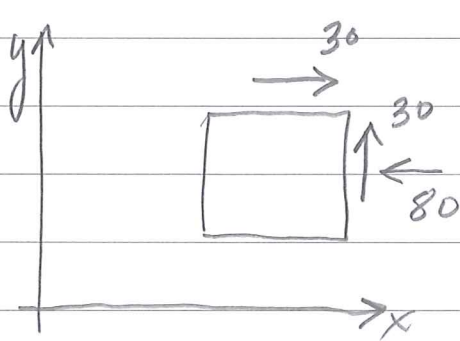
$$M_A = -\frac{1}{2} M_B = -\frac{QL}{34}$$

$$3) \Rightarrow V_C = \frac{1}{L} \left(\frac{QL}{2} - M_A \right) = Q \left(\frac{1}{2} + \frac{1}{34} \right) = \frac{18}{34} Q$$

$$2) \Rightarrow V_A = \frac{16}{34} Q$$

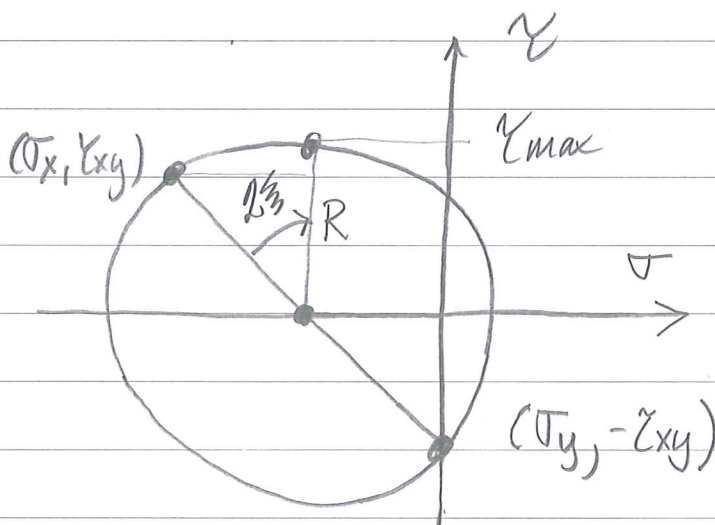
72 Hållfasthetslära TNE017 2015-04-14

4) Använd Mohrs cirkel LB p 174-175



ie $\sigma_x = -80$, $\sigma_y = 0$, $\tau_{xy} = 30$ MPa

Rita Mohrs cirkel:



Radie i Mohrs cirkel

$R = \tau_{max}$

$$R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\tau_{max} = \sqrt{\left(\frac{-80 - 0}{2}\right)^2 + 30^2} = 50 \text{ MPa}$$

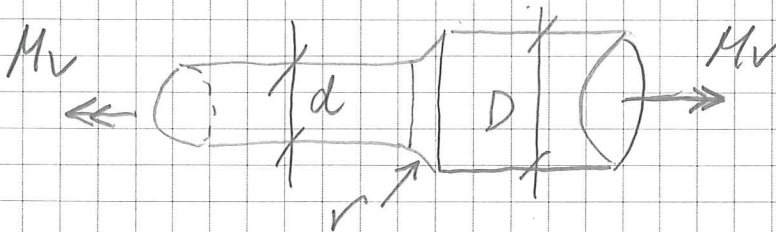
$$\tan 2\xi = \frac{\tau_{xy} - \left(\frac{\sigma_x + \sigma_y}{2}\right)}{\sigma_x - \left(\frac{\sigma_x + \sigma_y}{2}\right)} = \frac{\tau_{xy} - \sigma_y}{2\tau_{xy}} = -\frac{80}{60}$$

$$\rightarrow \varphi = \frac{1}{2}\xi = \frac{1}{2} \arctan\left(-\frac{4}{3}\right) = -28^\circ$$



22 Hållfasthetslära TME017 2015-04-14

5) Använd LB p 53, 57 & 251-252



$$\gamma_{\max} = k_t \gamma_{\text{nom}}$$

$$\gamma_{\text{nom}} = \frac{M_v}{W_v} = \frac{M_v \cdot 2}{\pi \left(\frac{d}{2}\right)^3}$$

Desutom $P = M_v \omega = M_v \frac{2\pi n}{60}$

Här är $\left. \begin{aligned} \frac{D}{d} &= \frac{160}{80} = 2 \\ \frac{r}{d} &= \frac{4}{80} = 0.05 \end{aligned} \right\} \Rightarrow k_t = 1.17$

Med $\gamma_{\max} = \gamma_{\text{till}} \Rightarrow M_v = \frac{\gamma_{\text{till}}}{k_t} \cdot \frac{\pi d^3}{16} = 3,25 \cdot 10^3 \text{ Nm}$

$$P = \frac{\gamma_{\text{till}}}{k_t} \cdot \frac{\pi d^3}{16} \cdot \frac{2\pi n}{60} = 307 \text{ kW}$$

Öka radien $r = 8 \text{ mm} \Rightarrow$

$\left. \begin{aligned} \frac{D}{d} &= 2 \\ \frac{r}{d} &= \frac{8}{80} = 0.1 \end{aligned} \right\} \Rightarrow k_t = 1.45 \Rightarrow P = 385 \text{ kW}$