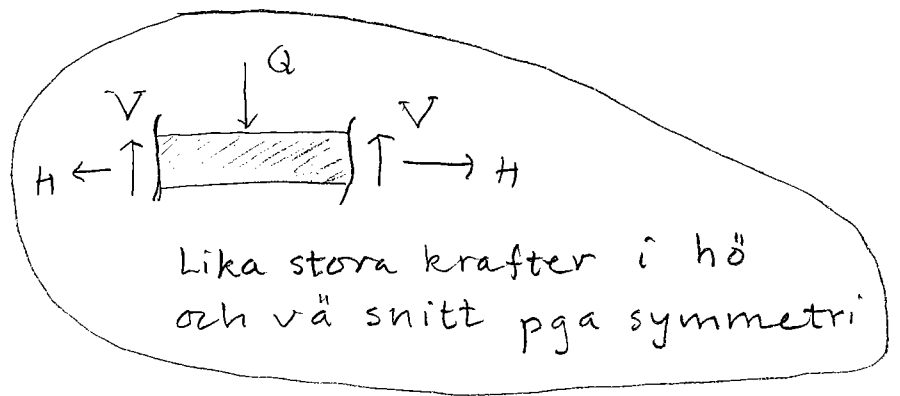
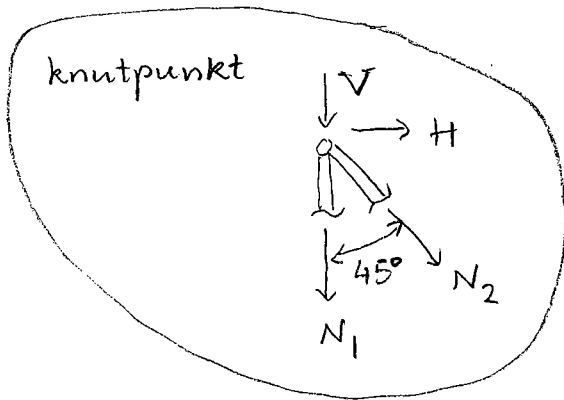


1) Friläggning:



$$\uparrow: 2V - Q = 0 \Rightarrow V = Q/2 \quad (1)$$

$$\uparrow: -V - N_1 - N_2/\sqrt{2} = 0 \quad (2)$$

$$\rightarrow: H + N_2/\sqrt{2} = 0 \quad (3)$$

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$$N_1 = \frac{Ea^2}{L} \delta_1 \quad N_2 = \frac{Ea^2}{\sqrt{2}L} \delta_2 \quad (4)$$

Kompatibilitet:

knutpunkt förflyttas i vertikalled pga symmetri och stel skiva, dvs Δ neråt antas



ur figur:
$$\begin{cases} \delta_1 = -\Delta \\ \delta_2 = -\Delta/\sqrt{2} \end{cases} \Rightarrow \delta_1 = \delta_2 \sqrt{2} \quad (5)$$

$$(4); (5) \Rightarrow N_1 = \sqrt{2} N_2 \sqrt{2} = 2N_2 \quad (5')$$

$$(5'), (1); (2) \Rightarrow -Q/2 - 2N_2 - N_2/\sqrt{2} = 0 \Rightarrow$$

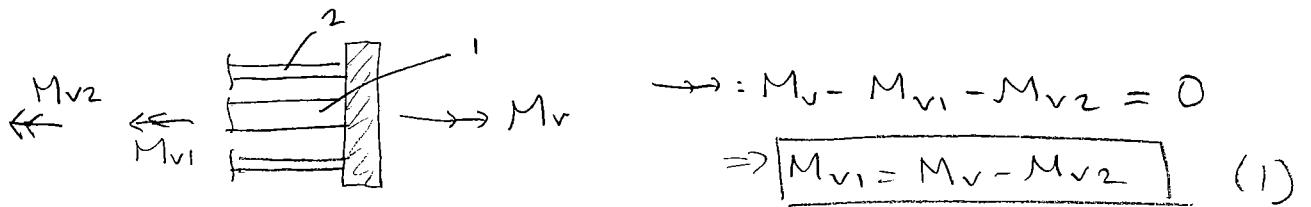
$$N_2 = \frac{-Q}{2(2 + 1/\sqrt{2})} \quad (2') \quad \stackrel{(5')}{\Rightarrow} N_1 = \frac{-Q}{2 + 1/\sqrt{2}}$$

\therefore Spänningen i de vertikala stängerna
diagonala — " —

$$\sigma_1 = N_1/a^2 \approx -18,5 \text{ [MPa]}$$

$$\sigma_2 = N_2/a^2 \approx -9,2 \text{ [MPa]}$$

2) Friläggning av stel skruva:



Kompatibilitet

vridningen av axlarna lika stor

$$\boxed{\varphi_1 = \varphi_2} \quad (2)$$

Konstitutiva samband

$$\varphi_1 = \frac{M_{v1} L}{G K_1} \quad \varphi_2 = \frac{M_{v2} L}{G K_2} \quad (3)$$

$$(3) ; (2) \Rightarrow M_{v1} = M_{v2} \frac{K_1}{K_2} \quad (2')$$

$$(2') ; (1) \Rightarrow M_{v2} \frac{K_1}{K_2} = M_v - M_{v2} \Rightarrow M_{v2} = \frac{M_v K_2}{K_1 + K_2} \quad (1')$$

$$(1') ; (1) \Rightarrow M_{v1} = \frac{M_v K_1}{K_1 + K_2} \quad (1'')$$

$$\text{Total vridning} = \varphi_1 = \varphi_2 \stackrel{(3)}{=} \frac{M_v L}{G (K_1 + K_2)} \quad (3')$$

$$\text{där} \quad K_1 = \frac{\pi}{32} \left[D^4 - (3D/5)^4 \right] = \frac{\pi D^4 17}{625} \approx 8,55 \cdot 10^6 \text{ [mm}^4 \text{]}$$

$$K_2 = \frac{4 A^2}{\int \frac{ds}{h}} = \frac{4 \cdot (5D/2)^4}{\frac{1}{t} \cdot 5D/2 \cdot 4} = \frac{125}{8} D^3 t \approx 3,13 \cdot 10^7 \text{ [mm}^4 \text{]}$$

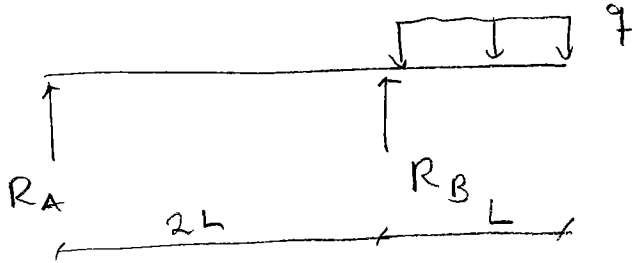
$$\Rightarrow \varphi_1 = \varphi_2 \approx 0,031 \text{ [rad]} \approx 1,8^\circ //$$

Max vridskjuvspänning

$$\tau_{\max,1} = \frac{M_{v1}}{W_{v1}} \stackrel{(6.14)}{=} \frac{M_{v1} D/2}{K_1} \stackrel{(1'')}{=} \frac{M_v D}{2(K_1 + K_2)} \approx 126 \text{ [MPa]}$$

$$\begin{aligned} \tau_{\max,2} &= \frac{M_{v2}}{W_{v2}} \stackrel{(6.69)}{=} \frac{M_{v2}}{2 A t} \stackrel{(1')}{=} \frac{M_v K_2}{2 (5D/2)^2 t \cdot (K_1 + K_2)} = \\ &\approx 314 \text{ [MPa]} \end{aligned}$$

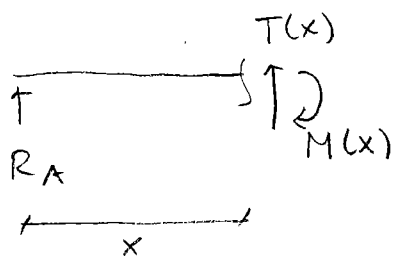
3) Friläggning av hela balken (statiskt bestämd)



$$\begin{cases} \uparrow: R_A + R_B - qL = 0 \\ \curvearrowright: -R_B \cdot 2L + qL(2L + L/2) = 0 \end{cases} \Rightarrow \begin{cases} R_B = 5qL/4 \\ R_A = -qL/4 \end{cases} \quad (1)$$

a) Snitta och frilägg

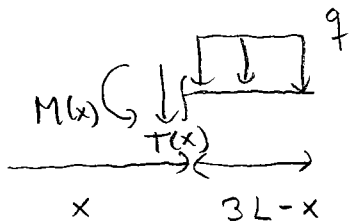
$0 < x < 2L$:



$$\uparrow: T(x) + R_A = 0 \Rightarrow T(x) = -R_A$$

$$\curvearrowright: M(x) + R_A x = 0 \Rightarrow M(x) = -R_A \cdot x$$

$2L < x < 3L$:

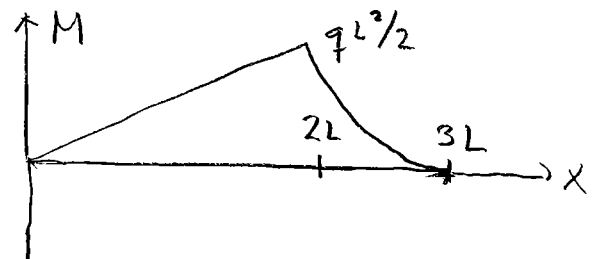
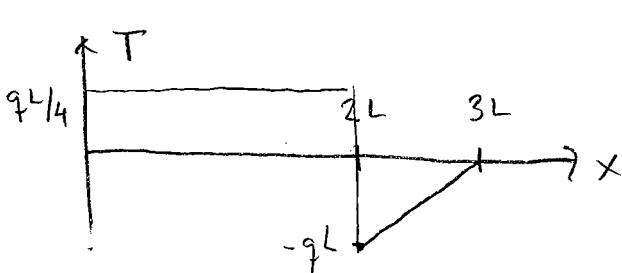


$$\uparrow: -q \cdot (3L - x) - T(x) = 0$$

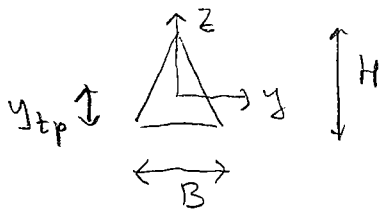
$$\Rightarrow T(x) = q \cdot (x - 3L)$$

$$\curvearrowright: -M(x) + q \cdot (3L - x) \cdot \frac{3L - x}{2} = 0$$

$$\Rightarrow M(x) = \frac{q}{2} (x - 3L)^2$$



b) Max böjnormalspänning σ_{\max} :



tp:s position $y_{tp} = H/3$

FS ger yttröghetsmomentet

$$I_y = BH^3/36$$

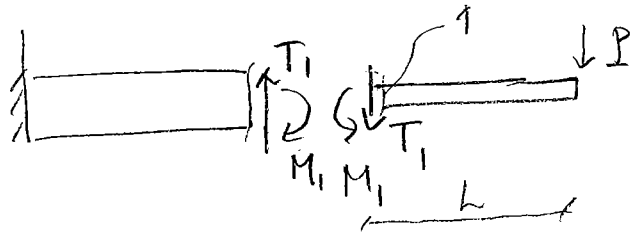
$$\sigma_{\max} = \frac{\max_x(|M(x)|) \cdot \max(|z|)}{I_y} = \frac{qL^2/2 \cdot 2H/3}{BH^3/36} =$$

$$= \frac{qL^2}{BH^2} \cdot 12$$

Max tillåten spänning $\sigma_{\text{till}} = 400 \text{ MPa}$

$$\Rightarrow q \leq \frac{BH^2}{L^2} \frac{1}{12} \cdot \sigma_{\text{till}} \approx 0,33 \cdot 10^3 \text{ [N/mm]} //$$

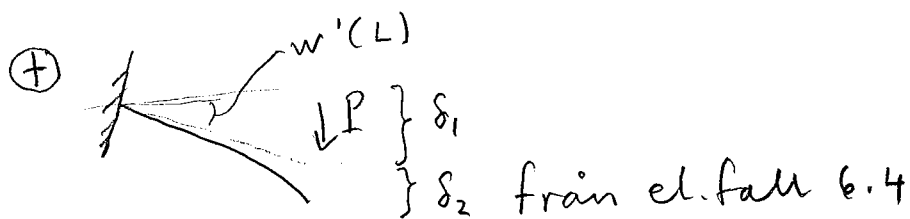
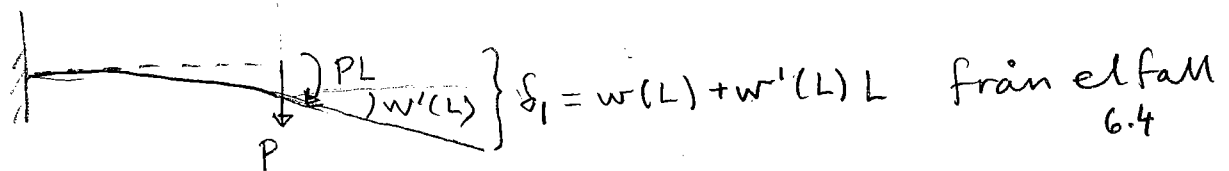
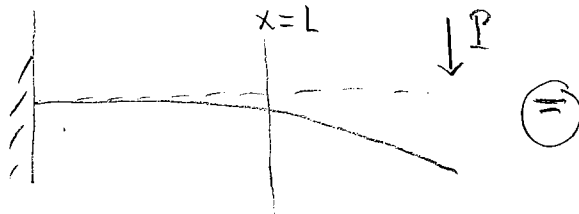
4) Snitta och frilägg



$$\uparrow: -T_1 - P = 0 \Rightarrow T_1 = -P$$

$$\curvearrowright: -M_1 + P \cdot L = 0 \Rightarrow M_1 = P \cdot L$$

Använd superposition av elementarfall för att bestämma nedböjning i höjande

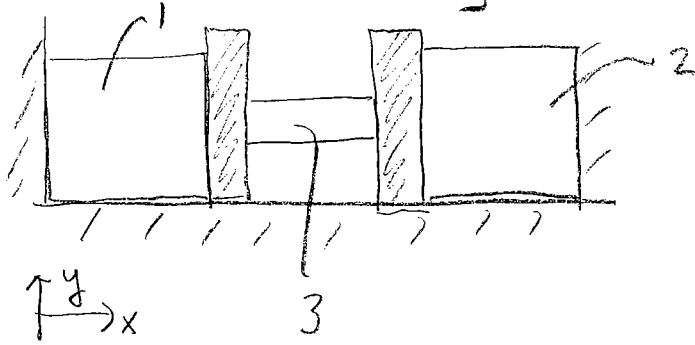


Totalt blir nedböjningen $\delta_1 + \delta_2 =$

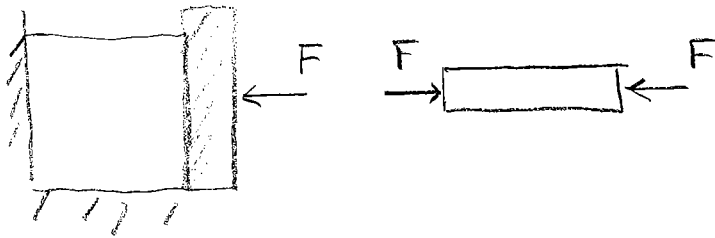
$$= \underbrace{\left(\frac{P \cdot L^3}{3 \cdot (2EI)} + \frac{P \cdot L \cdot L^2}{2 \cdot (2EI)} \right)}_{w(L)} + \underbrace{\left(\frac{P \cdot L^2}{2 \cdot (2EI)} + \frac{P \cdot L \cdot L}{(2EI)} \right)}_{w'(L)} \cdot L + \underbrace{\frac{P L^3}{3 \cdot EI}}_{\delta_2} =$$

$$= \frac{P L^3}{EI} \left(\frac{1}{6} + \frac{1}{4} + \frac{1}{4} + \frac{1}{2} + \frac{1}{3} \right) = \frac{P L^3}{EI} \frac{3}{2} //$$

5) Efter inplacering av stängen



Snitta och frilägg



$$\Rightarrow \begin{cases} \sigma_{x1} = -\frac{F}{L^2} = \sigma_{x2} & (1) \end{cases}$$

$$\begin{cases} \sigma_{x3} = -\frac{F}{\beta L^2} = \frac{1}{\beta} \sigma_{x1} & (2) \end{cases}$$

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Enaxlig spänning i alla tre delarna

$$\begin{cases} \epsilon_{x1} = \epsilon_{x2} = \frac{1}{E} \sigma_{x1} \stackrel{(1)}{=} -\frac{F}{E \cdot L^2} & (3) \end{cases}$$

$$\begin{cases} \epsilon_{x3} = \frac{1}{E} \sigma_{x3} \stackrel{(2)}{=} -\frac{F}{\beta E L^2} & (4) \end{cases}$$

Kompatibilitet

stängens längd efter inplacering
 $a + \delta_3$

klossarnas bredd —————

$$L + \delta_1, \quad L + \delta_2 = L + \delta_1$$

Eftersom avståndet mellan de stela väggarna är lika före och efter inplaceringen fås:

$$\underbrace{L + L + a - \Delta}_{\text{före}} = \underbrace{L + \delta_1 + L + \delta_1 + a + \delta_3}_{\text{efter}}$$

$$\Rightarrow -\Delta = 2 \cdot \delta_1 + \delta_3 \quad (5)$$

$$(3), (4) \text{ i } (5) \Rightarrow -\Delta = -\frac{2F}{EL} - \frac{Fa}{\beta EL^2} \Rightarrow F = \frac{\Delta EL}{2 + \frac{a}{\beta L}} \quad (5')$$

Nu kan töjningen i y-led bestämmas mha Hookes lag:

$$\begin{aligned} \epsilon_{y2} = \epsilon_{y1} &= -\frac{\nu}{E} \sigma_{x2} \stackrel{(1)}{=} \frac{\nu}{E} \frac{F}{L^2} \stackrel{(5')}{=} \frac{\nu}{E} \frac{\Delta EL}{L^2 (2 + \frac{a}{\beta L})} = \\ &= \frac{\nu \Delta}{2L + a/\beta} \end{aligned}$$

Höjdnöjningen fås nu som $\epsilon_{y2} \cdot L = \frac{\nu \cdot \Delta \cdot L}{2L + a/\beta} //$